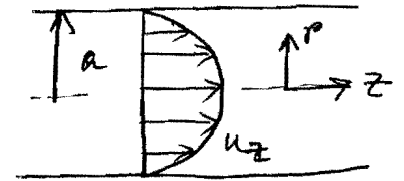
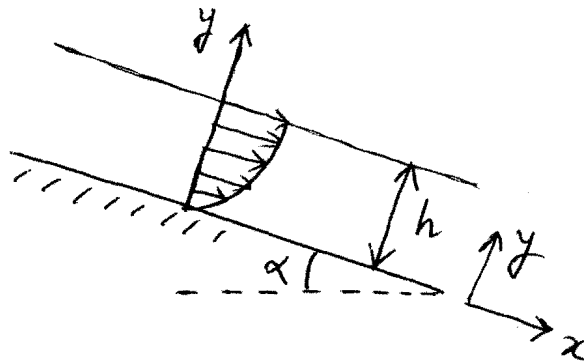


MAE 4230/5230
SOLUTION TO HOMEWORK 4

<2> Relation between flow under gravity down an inclined plane and 1-D pipe flow



$$u = \frac{g}{2\nu} y(2h-y) \sin\alpha$$

$$v = w = 0$$

$$u_z = \frac{P}{4\mu} (a^2 - r^2)$$

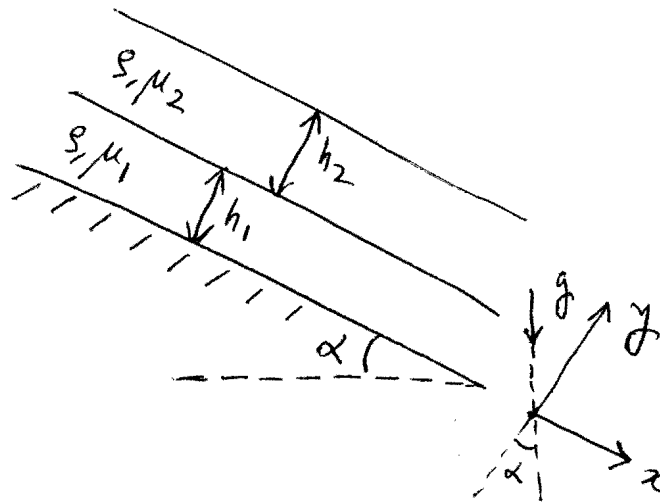
$$u_r = u_\theta = 0$$

1. $u = u_{max}$ at $y = h$
2. $\tau = \mu \frac{du}{dy} = 0$ at $y = h$

1. $u_z = u_{z,max}$ at $r = 0$
2. $\tau_{zr} = \mu \left(\frac{\partial u_z}{\partial r} + \frac{\partial u_r}{\partial z} \right)$
 $= \mu \frac{du_z}{dr} = 0$ at $r = 0$

Therefore, the flow down an inclined plane, viewed in the x - y plane, resembles half of a 1-D pipe flow (viewed in the r - z plane). In addition, both flows are steady, 1-D, incompressible and fully-developed. However, 1-D pipe flow is driven by an external pressure gradient while gravity drives the flow down an inclined plane.

<3> (2.4 in Acheson)



Steady, fully developed, 1-D, incompressible flow

Here, we have two fluids and we have to go back and forth between the lower and upper fluids to determine our unknown constants.

Lower fluid : $\underline{u} = [u_1(y), 0, 0]$

Navier-stokes equations under the conditions of the problem reduce to :

$$x\text{-component} : 0 = -\frac{1}{\rho} \frac{\partial p_1}{\partial x} + \nu_1 \frac{\partial^2 u_1}{\partial y^2} + g \sin \alpha$$

$$y\text{-component} : 0 = -\frac{1}{\rho} \frac{\partial p_1}{\partial y} + 0 - g \cos \alpha$$

$$z\text{-component} : 0 = -\frac{1}{\rho} \frac{\partial p_1}{\partial z} + 0 + 0$$

$$\therefore p_1 = p_1(x, y)$$

$$\therefore \frac{\partial p_1}{\partial y} = -\rho g \cos \alpha \Rightarrow \underline{p_1 = -(\rho g \cos \alpha) y + C_1(x)}$$

↳ ①

Upper fluid : $\underline{u} = [u_2(y), 0, 0]$

Navier-stokes equations under the conditions of the problem reduce to :

x-component : $0 = -\frac{1}{\rho} \frac{\partial p_2}{\partial x} + \nu_2 \frac{\partial^2 u_2}{\partial y^2} + g \sin \alpha$

y-component : $0 = -\frac{1}{\rho} \frac{\partial p_2}{\partial y} + 0 - g \cos \alpha$

z-component : $0 = -\frac{1}{\rho} \frac{\partial p_2}{\partial z} + 0 + 0$

$\therefore p_2 = p_2(x, y)$

$\therefore \frac{\partial p_2}{\partial y} = -\rho g \cos \alpha \Rightarrow p_2 = -(\rho g \cos \alpha) y + C_2(x)$

B.C. : at $y = h_1 + h_2$, $p_2 = p_0$ (atmospheric pressure)

$\Rightarrow p_0 = -\rho g \cos \alpha (h_1 + h_2) + C_2(x)$

$\therefore p_2 = p_2(y) = p_0 + \rho g \cos \alpha (h_1 + h_2 - y)$ — (2)

$\therefore \frac{\partial p_2}{\partial x} = 0 \Rightarrow$ flow is driven by gravity only down the inclined plane

$\therefore \frac{\partial^2 u_2}{\partial y^2} = -\frac{g \sin \alpha}{\nu_2}$

$\Rightarrow \frac{\partial u_2}{\partial y} = -\frac{g \sin \alpha}{\nu_2} y + C_5$ — (6)

B.C. : at $y = h_1 + h_2$, $\mu_2 \frac{du_2}{dy} = 0$

(zero stress at the free surface)

$$\Rightarrow -\frac{\rho g \sin \alpha}{\nu_2} (h_1 + h_2) + C_5 = 0$$

$$\Rightarrow C_5 = \frac{\rho g \sin \alpha}{\nu_2} (h_1 + h_2)$$

$$\therefore \frac{du_2}{dy} = \frac{\rho g \sin \alpha}{\nu_2} (h_1 + h_2 - y) \quad \text{--- (7)}$$

Lower fluid (Contd.) :

From upper fluid analysis [Equation (2)], we know $P_2(y)$

B.C. : at $y = h_1$, $P_1(h_1) = P_2(h_1)$ [Pressure continuity at the interface]

$$\Rightarrow -\rho g \cos \alpha (h_1) + C_1(x) = P_0 + \rho g \cos \alpha (h_2)$$

$$\Rightarrow C_1(x) = P_0 + \rho g \cos \alpha (h_1 + h_2)$$

$$\therefore P_1 = P_1(y) = P_0 + \rho g \cos \alpha (h_1 + h_2 - y) \quad \text{--- (3)}$$

From (3), $\frac{\partial P_1}{\partial x} = 0$ i.e., the flow is driven by gravity down the incline.

$$\therefore \frac{\partial^2 u_1}{\partial y^2} = -\frac{\rho g \sin \alpha}{\mu}$$

$$\Rightarrow \frac{\partial u_1}{\partial y} = -\frac{\rho g \sin \alpha}{\mu} y + C_3 \quad \text{--- (4)}$$

$$\therefore u_1 = -\frac{\rho g \sin \alpha}{2\mu} \left(\frac{y^2}{2}\right) + C_3 y + C_4$$

B.C. : at $y = 0$, $u_1 = 0$

$$\Rightarrow 0 = C_4$$

$$\therefore u_1 = -\frac{\rho g \sin \alpha}{2\mu} \left(\frac{y^2}{2}\right) + C_3 y \quad \text{--- (5)}$$

B.C.: at $y = h_1$, $\mu_1 \frac{du_1}{dy} = \mu_2 \frac{du_2}{dy}$ [Newton's third law; shear stress balance at the interface]

Using equation (7) from upper fluid analysis,

$$-\mu_1 \frac{g \sin \alpha}{\nu_1} y + \mu_1 c_3 = \mu_2 \frac{g \sin \alpha}{\nu_2} \cdot (h_1 + h_2 - y)$$

$$\Rightarrow -\rho g \sin \alpha (h_1) + \mu_1 c_3 = \rho g \sin \alpha (h_2)$$

$$\Rightarrow c_3 = \frac{g \sin \alpha}{\nu_1} (h_1 + h_2)$$

Putting in (5), we get the velocity of the lower fluid as

$$u_1 = -\frac{g \sin \alpha}{\nu_1} \left(\frac{y^2}{2} \right) + \frac{g \sin \alpha}{\nu_1} (h_1 + h_2) y$$

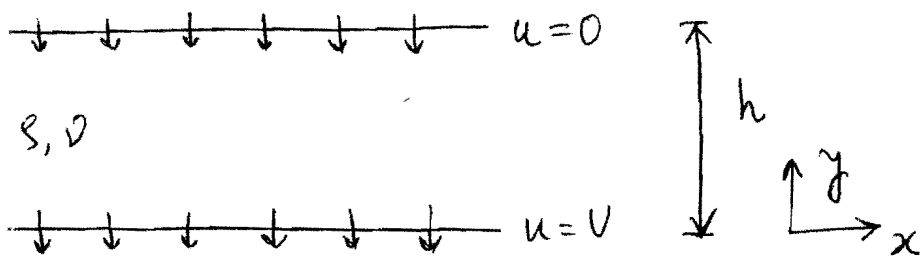
$$\Rightarrow \boxed{u_1(y) = [(h_1 + h_2) y - \frac{1}{2} y^2] \frac{g \sin \alpha}{\nu_1}}$$

* In the above analysis, we find that μ_2 falls out of the expression for u_1 when we apply the shear stress balance at the interface. Consider the situation when we only have the lower fluid (μ_1). Then, at the free surface $y = h_1$, $\mu_1 \frac{\partial u_1}{\partial y} = 0$. Now, let us ~~put~~ introduce

the upper fluid (μ_2) on top. Now, at $y = h_1$, $\mu_1 \frac{\partial u_1}{\partial y}$ is no longer zero but modified to be equal to $\mu_2 \frac{\partial u_2}{\partial y}$, where u_2 now is such that at $y = h_1 + h_2$, $\mu_2 \frac{\partial u_2}{\partial y} = 0$. Thus, it turns out that u_1 does not depend on μ_2 due to the balance of stresses at the interface but depends on h_2 due to the stress-free boundary condition at the free surface (which can be thought of as being shifted from $y = h_1$ to $y = h_1 + h_2$ by addition of the fluid on top).

* Another way to look at it: The shear stress at the interface on the lower fluid is the result of a component of the weight of the upper fluid, which depends on the volume of the upper fluid (and hence on h_2) but not on μ_2 . That is why, we find that $\mu_2 \frac{\partial u_2}{\partial y}$ at $y = h_1$ is independent of μ_2 but dependent on h_2 . Note that u_1 could potentially depend on ρ_2 ($= \rho_1$ here), which supports this argument.

<4> (2.6 in Acheson)



Steady, fully developed, 2-D, incompressible flow

$$\underline{u} = [u(y), v(y), 0]$$

Continuity: $\nabla \cdot \underline{u} = 0 \Rightarrow \frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} = 0$

$$\Rightarrow \frac{\partial v}{\partial y} = 0$$

$$\Rightarrow v \neq v(y) = \text{constant}$$

But, at $y=0$, $v = -v_0$

$\therefore \boxed{v = -v_0}$ for all y i.e. throughout the channel

$$\therefore \underline{u} = [u(y), -v_0, 0]$$

Momentum (Navier-Stokes):

$$\frac{\partial \underline{u}}{\partial t} + (\underline{u} \cdot \nabla) \underline{u} = -\frac{1}{\rho} \nabla P + \nu \nabla^2 \underline{u}$$

x-component: $\frac{\partial u}{\partial t} + u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y} = -\frac{1}{\rho} \frac{\partial P}{\partial x} + \nu \left(\frac{\partial^2 u}{\partial x^2} + \frac{\partial^2 u}{\partial y^2} \right)$

$$\Rightarrow -v_0 \frac{\partial u}{\partial y} = -\frac{1}{\rho} \frac{\partial P}{\partial x} + \nu \frac{\partial^2 u}{\partial y^2} \quad \text{--- ①}$$

Here, the flow is driven by the bottom plate moving with a velocity $u = U$. Therefore, we can assume no pressure gradient i.e. $\frac{\partial P}{\partial x} = 0$

\therefore Equation (1) reduces to

$$\boxed{\frac{d^2 u}{dy^2} + \left(\frac{v_0}{\nu}\right) \frac{du}{dy} = 0} \quad \text{--- (2)}$$

Equation (2) is a 2nd order linear ODE with constant coefficients. Hence, we try the solution $u(y) = e^{ry}$, where r is a constant.

Putting in (2), $r^2 e^{ry} + \left(\frac{v_0}{\nu}\right) r e^{ry} = 0$

$$\Rightarrow r^2 + \left(\frac{v_0}{\nu}\right) r = 0$$

$$\Rightarrow r \left[r + \frac{v_0}{\nu} \right] = 0$$

$$\therefore r_1 = 0; r_2 = -\left(\frac{v_0}{\nu}\right)$$

\therefore The general solution is given by :

$$\begin{aligned} u(y) &= c_1 e^{r_1 y} + c_2 e^{r_2 y} \\ &= c_1 + c_2 \exp\left(-\frac{v_0}{\nu} y\right) \end{aligned}$$

Boundary conditions :

(i) at $y=0$, $u=U \Rightarrow \underline{U = c_1 + c_2}$ --- (3)

$$(ii) \text{ at } y=h, u=0 \Rightarrow 0 = c_1 + c_2 \exp\left(-\frac{v_0}{\nu} h\right)$$

$$\therefore \textcircled{3} - \textcircled{4} \text{ gives, } U = c_2 \left[1 - \exp\left(-\frac{v_0}{\nu} h\right)\right]$$

$$\Rightarrow c_2 = \frac{U}{1 - \exp\left(-\frac{v_0}{\nu} h\right)}$$

$$\therefore c_1 = -c_2 \exp\left(-\frac{v_0}{\nu} h\right) = -\frac{U \exp\left(-\frac{v_0}{\nu} h\right)}{1 - \exp\left(-\frac{v_0}{\nu} h\right)}$$

$$\therefore u(y) = c_1 + c_2 \exp\left(-\frac{v_0}{\nu} y\right)$$

$$= \frac{-U \exp\left(-\frac{v_0}{\nu} h\right) + U \exp\left(-\frac{v_0}{\nu} y\right)}{1 - \exp\left(-\frac{v_0}{\nu} h\right)}$$

$$= U \left[\frac{\exp\left(-\frac{v_0}{\nu} y\right) - \exp\left(-\frac{v_0}{\nu} h\right)}{1 - \exp\left(-\frac{v_0}{\nu} h\right)} \right]$$

\therefore The velocity field is given by

$$\underline{u} = \left[U \left\{ \frac{\exp\left(-\frac{v_0}{\nu} y\right) - \exp\left(-\frac{v_0}{\nu} h\right)}{1 - \exp\left(-\frac{v_0}{\nu} h\right)} \right\}, -v_0, 0 \right]$$

If the velocity profile within the channel is to be similar to a boundary layer, then for some $y < h$, the velocity u should reduce to 1% of the boundary velocity U .

Let $\frac{v_0 h}{\nu} = M \gg 1$ and let us now find y for which $u = 1\%$ of $U = 0.01 U$

$$\therefore 0.01 U = U \left[\frac{\exp\left[-\frac{v_0}{\nu} y\right] - \exp[-M]}{1 - \exp[-M]} \right]$$

$$\Rightarrow 0.01 = \exp\left(-\frac{v_0}{\nu} y\right)$$

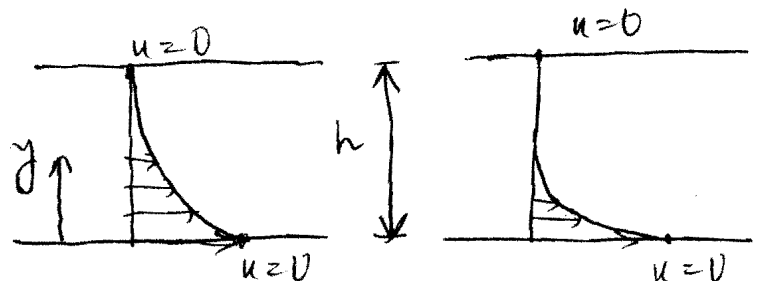
$$\Rightarrow \ln(0.01) = -\frac{v_0}{\nu} y$$

$$\Rightarrow y = \frac{4.605 \nu}{v_0} = \left(\frac{4.605}{M}\right) h$$

\therefore If $M \gg 1$, $y \ll h$. Also, $y \propto \nu$. Therefore,

we conclude that if $\frac{v_0 h}{\nu} \gg 1$, the velocity profile in the channel resembles a boundary layer with thickness proportional to the kinematic viscosity ν . Also, larger the value

of $\frac{v_0 h}{\nu}$, thinner is the boundary layer.



increasing
 $\frac{v_0 h}{\nu}$